

1 **Three-dimensional Linear Eddy Modeling of a**
2 **Turbulent Lifted Hydrogen Jet Flame in a Vitiated**
3 **Co-flow**

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8 **Abstract** A new methodology for modeling and simulation of reactive flows is
9 reported in which a 3D formulation of the Linear Eddy Model (LEM3D) is used
10 as a post-processing tool for an initial RANS simulation. In this hybrid approach,
11 LEM3D complements RANS with unsteadiness and small-scale resolution in a
12 computationally efficient manner. To demonstrate the RANS-LEM3D model, the
13 hybrid model is applied to a lifted turbulent N₂-diluted hydrogen jet flame in
14 a vitiated co-flow of hot products from lean H₂/air combustion. In the present
15 modeling approach, mean-flow information from RANS provides model input to
16 LEM3D, which returns the scalar statistics needed for more accurate mixing and
17 reaction calculations. Flame lift-off heights and flame structure are investigated

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18 in detail, along with other characteristics not available from RANS alone, such as
19 the instantaneous and detailed species profiles and small-scale mixing.

20 **Keywords** Linear Eddy Model · Turbulent mixing · Subgrid scalar closure ·

21 Turbulent reactive flows

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1 Introduction

State-of-the-art simulation tools in industrial applications are mainly based on the Reynolds-Averaged Navier-Stokes (RANS) equations, and hence lack the spatial and temporal resolution provided by large eddy simulation (LES) or direct numerical simulation (DNS). While DNS can give detailed insight into flow structures and turbulence-flame interactions, the method is, currently and in the foreseeable future, out of reach for most practical applications. LES and RANS, however, rely on the gradient diffusion model with the counter-gradient assumption.¹ But where LES models the smallest scales (generally assumed to be isotropic), RANS provides no information. Small-scale resolution, however, is needed to give accurate predictions of the mixing and chemistry in turbulent combustion processes. Due to the computational cost associated with DNS and LES, alternative methods to provide small-scale resolution have been pursued in recent years. One-dimensional approaches, such as the Linear Eddy Model (LEM) [1, 2] and the One-Dimensional Turbulence (ODT) model [3], are methods that resolve all scales of turbulent reactive flows at a computationally affordable cost and with promising results [4, 5, 6]. In the present study, we employ a novel formulation called the 3-dimensional Linear Eddy Model (LEM3D) [7, 8], recently implemented with detailed chemistry, to investigate the lift-off height, the flame structure and other characteristics of a turbulent lifted hydrogen jet flame in a hot vitiated co-flow.

LEM3D is developed as a research tool, both in order to complement the capabilities of RANS or LES by resolving the flame structure and to improve predictions of turbulent reactive flows. While RANS gives no information other than the averaged field and LES makes use of a sub-grid model to get information about the small scale resolution, LEM3D makes use of the averaged mass-fluxes and turbulent flow field to emulate the behavior of turbulent eddies down to the smallest scales through stochastic events called triplet maps. The formulation is

¹ For non-reacting flows the counter gradient assumption implies that the averaged transport $\rho \mathbf{u}'' \phi''$ of a scalar ϕ is oriented in a direction opposite to the normal gradient of the turbulent diffusion.

50 a 3D construction based on LEM, involving three orthogonally intersecting ar-
51 rays of 1D LEM domains, and coupled so as to capture the 3D character of fluid
52 trajectories. In the hybrid approach presented here, the averaged mass-fluxes and
53 turbulent flow field are obtained in RANS and fed to LEM3D as model input.

54 The vitiated co-flow burner, used as a demonstration case in the present study,
55 was developed at UC Berkeley and first presented by Cabra et al. [9, 10]. The
56 burner enables studies of flame lift-off and stabilization mechanisms in an envi-
57 ronment similar to that of a gas turbine combustor. The vitiated co-flow burner
58 and similar experiments have been used extensively for model validation in recent
59 years, e.g. a virtually identical experimental set-up was installed at the University
60 of Sydney with advanced diagnostics to probe the location and structure of au-
61 toignition kernels [11], conditional analysis by Cheng et al. [12] were used to reveal
62 the reaction zone structure in mixture fraction coordinates, and at UC Berkeley a
63 pressurized vitiated co-flow burner was installed in 2013 for investigation of the sta-
64 tistical likelihood of autoignition events in the mixing region [13]. Myhrvold et al.
65 [14] explored the sensitivity of predictions to the boundary conditions to validate
66 the Eddy Dissipation Concept, and the DQMOM based PDF transport modeling
67 by Lee et al. [15] was validated to indicate that the model has the capability of
68 predicting the autoignition, the flame lift-off and the stabilization process.

69 The hot co-flow of the vitiated co-flow burner consists of combustion products
70 from lean premixed hydrogen-air flames, which mimics the recirculated hot com-
71 bustion products in practical combustors to enhance flame stability. The advantage
72 is that the simplified flow of the burner removes the complexity of recirculating
73 flows and hence makes the vitiated co-flow burner attractive for numerical mod-
74 eling. The characteristics of autoignition and lift-off heights of turbulent H_2/N_2
75 flames issuing into hot co-flows of combustion products has been extensively stud-
76 ied by, e.g., Masri et al. [16] and Cao et al. [17] using PDF calculations. Myhrvold
77 et al. [14] conducted a series of simulations and indicated the extent to which
78 turbulence models influence the predicted lift-off height with Magnussen's Eddy

79 Dissipation Concept [18]. While Cao et al. [17] indicate that the lift-off is primar-
80 ily controlled by chemistry, later studies showed that the autoignition events in
81 unsteady flames are controlled by both chemistry and turbulent mixing e.g. [19].
82 A 3D DNS at Sandia National Laboratories simulating a planar hydrogen jet is-
83 suing with high velocity in hot slow air [20] seemed to put an end to the original
84 uncertainty expressed by Cabra et al. [9], that is, autoignition was identified as
85 the dominant stabilization mechanism for a lifted hydrogen flame in a hot co-flow
86 and thus more important than the effects of flame propagation.

87 This paper reports on a new methodology for combustion modelling and sim-
88 ulation in which LEM3D is extended to reactive flows and applied to the Berkeley
89 vitiated co-flow burner. The Berkeley burner has been selected since it is a challeng-
90 ing flame relevant for gas turbine applications. In Section 2 we present a summary
91 of the Linear Eddy Model, the LEM3D formulation, and the implementation of
92 chemistry into the model. Also, the details of the initial RANS simulation is pro-
93 vided. The results of the study are presented in Section 3, where scatter plots,
94 contour plots and axial profiles of various scalar quantities are given. Some con-
95 cluding remarks are contained in Section 4.

96 **2 Numerical Model and Setup**

97 **2.1 Linear Eddy Modeling**

98 The Linear Eddy Model developed by Kerstein [1, 2] was formulated to capture
99 the mixing and reaction of scalars (like chemical species) in a computationally
100 affordable manner. This is achieved by a reduced one-dimensional representation
101 of the scalar fields, for which all relevant length and time scales are fully resolved.
102 The basic idea is that the statistical description of the scalar fields in one dimension
103 is representative of the scalar statistics of the real 3-dimensional flow.

To give further motivation for the concepts of LEM modeling, consider first the general transport equation for a reactive scalar ϕ , written as

$$\frac{\partial \rho \phi}{\partial t} + \frac{\partial \rho u_j \phi}{\partial x_j} = \frac{\partial}{\partial x_j} \left[\rho D_M \frac{\partial \phi}{\partial x_j} \right] + \rho \omega_\phi, \quad (1)$$

104 where ρ is the density, u_j is the velocity component in the coordinate direction x_j ,
 105 D_M is the molecular diffusivity, and ω_ϕ is the chemical reaction rate. In the above
 106 equation a gradient type model is assumed for the diffusive flux (Fick's law), and
 107 the molecular diffusivity D_M is assumed to be represented by a mixture-averaged
 108 quantity.

For turbulent flows, the most common approximation is the Reynolds-averaged equation, below expressed in its most general form with Favre averaging, i.e.,

$$\frac{\partial \bar{\rho} \tilde{\phi}}{\partial t} + \frac{\partial \bar{\rho} \tilde{u}_j \tilde{\phi}}{\partial x_j} = \frac{\partial}{\partial x_j} \left[\bar{\rho} D_M \frac{\partial \tilde{\phi}}{\partial x_j} - \bar{\rho} \widetilde{u_j'' \phi''} \right] + \bar{\rho} \tilde{\omega}_\phi, \quad (2)$$

109 where $\bar{\rho}$ denotes the mean of ρ , $\tilde{\phi} = \overline{\rho \phi} / \bar{\rho}$ is the Favre-averaged scalar field, and
 110 $u_j'' = u_j - \tilde{u}_j$ is the fluctuation of u_j about the Favre average \tilde{u}_j . The term
 111 $\partial (\bar{\rho} \tilde{u}_j \tilde{\phi}) / \partial x_j$ gives the advective transport based on the velocity field \tilde{u}_j . The
 112 primary challenge of this approach is that it treats turbulent mixing, which by
 113 nature is an advective process, as a diffusion term through the mass-averaged
 114 scalar fluxes $\bar{\rho} \widetilde{u_j'' \phi''} = -\bar{\rho} D_T \frac{\partial \tilde{\phi}}{\partial x_j}$. This is called the gradient-diffusion assumption,
 115 where the turbulent diffusivity D_T is positive. The implication is that the scalar
 116 flux is in the opposite direction of the mean scalar gradient. In other words, the
 117 transport of a scalar is always in the direction from a region of higher mean scalar
 118 concentrations to a region of lower concentrations. However, for inhomogeneous,
 119 anisotropic or streamline turbulence this might not be the case, i.e., in these regions
 120 we could have counter-gradient diffusion which does not obey the gradient-diffusion
 121 assumption.

A unique feature of LEM is that the model in fact makes an explicit distinction between the processes of molecular and turbulent diffusion, i.e., turbulent mixing

is treated as an advective process. This feature is crucial in order to capture the dissimilar influences of these processes on the scalar mixing, and is achieved because all relevant scales of the turbulent flow is resolved. For the 1D LEM, the governing equation of scalar transport is expressed as

$$\frac{\partial \rho \phi}{\partial t} + TM = \frac{\partial}{\partial x} \left[\rho D_M \frac{\partial \phi}{\partial x} \right] + \rho \omega_\phi, \quad (3)$$

122 where the molecular diffusion $\frac{\partial}{\partial x} \left[\rho D_M \frac{\partial \phi}{\partial x} \right]$ and chemical reactions $\rho \omega_\phi$ are solved
 123 directly on the LEM domain, and TM denotes stochastic triplet maps (see Sec.
 124 2.2). The stochastic stirring and diffusive mixing affect the chemical reactions and
 125 the subsequent heat release. In terms of implementation, the reactive-diffusive
 126 processes are punctuated by the stochastic triplet map events TM .

127 In general, there is a governing transport equation (3) for each of the scalars
 128 (species, temperature, etc.) being part of a particular reactive flow field. Thus,
 129 LEM naturally accommodates for multiple species undergoing chemical reactions.
 130 In particular, LEM takes into account effects of differential diffusion, which plays
 131 an important role in hydrogen combustion [21, 22]. A full description of the one-
 132 dimensional LEM can be found in [1, 2].

133 2.2 The triplet map

134 The triplet maps are stochastic events in LEM which represent turbulent advection
 135 (stirring). The turbulent stirring is a distinct physical mechanism governing the
 136 mixing of scalar fields. In Lagrangian terminology, the triplet maps rearrange fluid
 137 cells, represented by the computational cells of the discretized one-dimensional
 138 domain, in such a manner that scalar length scales are reduced and local gradi-
 139 ents are magnified. This is in accordance with the effects of compressive strain
 140 in turbulent flow. These stochastic events hence emulate the effects of individual
 141 turbulent eddies on the scalar concentration fields as illustrated in Fig. 1. Note

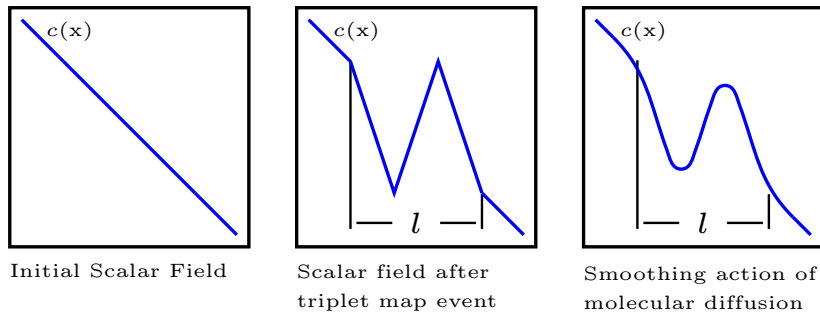


Fig. 1: Schematic diagram of a triplet mapping event of size l and the competing actions of molecular diffusion and reaction after a rearrangement event occurs.

142 that the effect of a single triplet map is limited to the section l , while the molecular
 143 diffusion generally affect the entire 1D domain.

144 2.3 LEM3D

145 LEM3D endeavours to maintain the distinction between chemical reactions, molec-
 146 ular diffusion and turbulent mixing, which means that the scalars do not mix at the
 147 molecular level by other processes than molecular diffusion. The LEM3D formu-
 148 lation, first described in [7, 8], incorporates three orthogonally intersecting arrays
 149 of 1D LEM domains, with intersecting LEM domains coupled in a Lagrangian
 150 sense by non-diffusive fluid-cell transfers from one domain to another (see Fig. 2).
 151 LEM3D thus provides small-scale resolution in all three spatial directions of the
 152 turbulent flow field, as well as time-resolved unsteadiness.

153 Diffusive time advancement takes place on each LEM domain in small sub-
 154 cycling steps within a coarser advective time step. The sub-cycling is punctuated
 155 by the randomly occurring stirring events, i.e., the triplet maps.

156 The coupling of the LEM domains is associated with the larger time step
 157 corresponding to the coarse-grained spatial scale defined by the intersections of
 158 orthogonal LEM domains. By construction, these intersections define a Cartesian
 159 mesh of cubic control volumes (3DCVs).

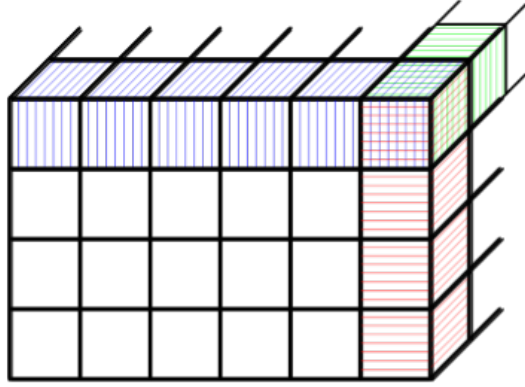


Fig. 2: The flow domain of the LEM3D simulation with the coarse Cartesian mesh consisting of $45 \times 45 \times 84$ grid cells. The superimposed fine-scale resolution is illustrated by the coloured LEM domains in red, blue and green. One domain is shown in each coordinate direction and they intersect in the top-front corner control volume (3DCV) in LEM3D. Note that the actual LEM resolutions used in the simulations are much higher than illustrated in the figure.

The governing equation follows the structure of the stand-alone 1D LEM, but now includes the advection term, i.e.,

$$\frac{\partial \rho \phi}{\partial t} + \frac{\partial \bar{\rho} \tilde{u}_\alpha \phi}{\partial x_\alpha} + TM_j = \frac{\partial}{\partial x_j} \left[\rho D_M \frac{\partial \phi}{\partial x_j} \right] + \rho \omega_\phi, \quad (4)$$

160 where the index j indicates that the terms are implemented on 1D LEM domains
 161 in three directions. Note that the conventional summation over the repeated index
 162 j is not implied for the right-hand-side term.

163 The averaged advection process $\partial(\bar{\rho} \tilde{u}_\alpha \phi)/\partial x_\alpha$ is governed by a velocity and
 164 mean density field $\bar{\rho}$ which are prescribed from a global flow solver or measure-
 165 ments. The advection is implemented deterministically by Lagrangian displace-
 166 ments of fluid cells. This process involves the intersection and coupling of the 1D
 167 domains. The other terms of Eq. (4) are explained in Section 2.1.

168 2.4 Implementation of chemistry

169 LEM3D may be considered as a "1D-DNS" in all three directions, i.e., the model
170 is resolved down to the Batchelor scale represented by the 1D LEM cells. Hence,
171 the 1D cells, called *wafers*, can be considered as homogeneous reactors which im-
172 plies that the chemistry is implemented directly in LEM3D. In previous work,
173 unity Lewis number, infinitely fast chemistry, and adiabatic conditions were im-
174 plemented [8]. Further, the chemistry was represented through a single conserved
175 scalar, i.e., the mixture fraction ξ . In the current formulation, detailed and finite
176 rate chemistry is implemented with the Li mechanism [23] and solved using the
177 CHEMKIN II software package. The chemical source term $\rho\omega_\phi$ of Eq. (4) is solved
178 directly through the stiff solver DVODE [24]. The individual diffusion coefficients
179 for the different species are implemented through the mixture-averaged diffusion
180 coefficient approach [25].

181 Thermal expansion, i.e., dilatation, was previously accounted for by creating
182 new cells in integer steps when the local wafer pressure was an integer number
183 higher than the surrounding pressure. In the new implementation this is accounted
184 for by increasing the cell volume and performing a regridding subsequently to every
185 diffusive-reactive time step.

186 It should be mentioned that a third way to account for thermal expansion was
187 suggested and implemented by Oevermann et al. in 2008 [26]. In that approach the
188 expansion induces a flow out of the fluid cell in an Eulerian manner. This option
189 causes some artificial diffusion. This is also the case in the modified implementation
190 of LEM3D, since the regridding forces fluid to cross the cell boundaries and mix
191 with the adjacent cells.

192 2.5 RANS simulation

193 The hybrid RANS-LEM3D approach is based on an initial RANS simulation which
194 provides mean-flow information in the form of input files to LEM3D. The 3D RANS

195 simulation is here performed using the ANSYS Fluent package, which solves the
196 Reynolds-Averaged Navier-Stokes equations for the mean conservation of mass,
197 momentum and energy, along with the k - ε turbulence model. The RANS simu-
198 lation is performed on a cuboidal $85 \times 85 \times 120$ grid using a modified k - ε model.
199 The jet inlet is approximated by a single grid cell such that the area of the jet
200 is preserved, i.e., the grid size Δx is given by $(\Delta x)^2 = \pi(d/2)^2$, where the jet
201 diameter is $d = 4.57$ mm. This coarse grid might seem as a crude approximation
202 but is chosen to demonstrate the potential of the hybrid model. Additional RANS
203 simulations with finer grids indicated that a grid-independent solution could be
204 attained with a Cartesian grid of the order of 10^3 more grid cells than the coarse
205 grid. An approximate measure of the error introduced by the coarse grid is that the
206 centerline axial mean velocity differ by about 12% on the average from the grid-
207 independent solution, while the jet velocity half-width is about 13% wider than
208 such a solution at the axial location of the lifted flame base at $10d$, as measured
209 by Cabra et al. [9]. Nonetheless, with the focus here on method demonstration,
210 the mean-flow information based on the coarse grid simulation is considered as
211 sufficiently accurate.

212 The numerical scheme used for the RANS simulation is given in Table 1. Note
213 that $C_{1\varepsilon}$ and $C_{2\varepsilon}$ were set in accordance with Myhrvold et al. [14] to correct for
214 the overestimated spreading rate by the standard k - ε model.

215 **Table 1:** Numerical conditions selected for computing the H_2/N_2 jet flame in a
vitiating co-flow.

Domain	Cuboid, 85×85×120
Solver	Steady state
Turbulence model	Modified k - ε with $C_\mu = 0.09$, $C_{1\varepsilon} = 1.44$, $C_{2\varepsilon} = 1.83$, $\sigma_k = 1$, $\sigma_\varepsilon = 1.3$
Turbulence-chemistry interaction	Eddy-Dissipation Concept
Discretization schemes	Standard for pressure SIMPLEC for pressure-velocity coupling Second order upwind for momentum and turbulent kinetic energy
Under-relaxation factors	Pressure= 0.3, Body forces= 0.9, Momentum= 0.7, Density= 0.9

217 The boundary conditions used in the computation are the same as those applied
 218 in the simulations by Cabra et al. [9] and Myhrvold et al. [14], and are detailed in
 219 Table 2.

220 **Table 2:** Flame and flow boundary conditions for the jet and the co-flow.

	Central jet	Co-flow
Volumetric flow of H ₂ [L _{STP} /min]	25	225
Volumetric flow of N ₂ [L _{STP} /min]	75	
Volumetric flow of air [L _{STP} /min]		2100
Temperature [K]	305	1045
Mean velocity [m/s]	107	3.5
Reynolds number	23600	18600
Diameter [m], d	0.00457	0.21
Mean mole fraction, H ₂	0.2537	0.0005
Mean mole fraction, N ₂	0.7427	0.7532
Mean mole fraction, O ₂	0.0021	0.1474
Mean mole fraction, H ₂ O	0.0015	0.0989

222 With the given numerical scheme and the boundary and initial conditions,
223 the RANS simulation resulted in a close-to-attached flame with a lift-off height
224 of only $1.4d$. Cabra et al. [9] found, through measurements, that the actual lift-
225 off height was $10d$. The OH contour was used to determine the lift-off height,
226 where the lift-off is defined as the axial location at which the OH mass fraction
227 first reaches 600 ppm as in [9, 10, 14]. The challenge with the turbulent lifted
228 jet flame is the high sensitivity of the lift-off height to a variety of factors, such
229 as the co-flow temperature and the precise dilution level of the fuel jet. Thus,
230 a series of RANS simulations with different combinations of the Energy Prandtl
231 number and the turbulent Schmidt number away from the Fluent default values
232 showed that converged flames with just about any lift-off height could be attained.
233 Moreover, during these RANS simulations issues were encountered with respect
234 to flame stabilization. This seemed to be due to hysteresis effects. Hysteresis on
235 $T_{\text{co-flow}}$, V_{jet} and $y_{\text{N}_2, \text{jet}}$ affects the stability regimes layout, though for the viti-
236 ated co-flow burner, stability is most sensitive to $y_{\text{N}_2, \text{jet}}$, i.e., the dilution level.
237 These hysteresis effects influencing the transition to the lifted condition are well
238 known and documented [27]. However, it is reported for a lifted flame with similar
239 conditions that the hysteresis effect will not affect the stability boundaries in the
240 unsteady regime [13].

241 Since the intention here is to use the vitiated co-flow burner as a demonstration
242 case for the hybrid RANS-LEM3D model, the original RANS simulation with the
243 close-to-attached flame was used as input for the subsequent LEM3D simulation.
244 One aspect of this is to test whether LEM3D with the given flow field can correct
245 for the missing lift-off compared to the experiment. In other words, the sensitivity
246 of the model with respect to the flow field is probed.

247 2.6 The hybrid RANS-LEM3D model

248 The hybrid model presented in this paper is based on an initial RANS simulation
249 in the Fluent flow solver which in turn generates the necessary model input to

Table 3: LEM3D input properties

Δx	4.05×10^{-3} m
Δt	1.25×10^{-6} s
Δx_w	4.05×10^{-5} m
σ_k	0.7
Pressure	1 bar
Advective CFL # RANS	0.1
LEM resolution	100

250 LEM3D. The RANS model input to LEM3D is mean-flow information such as
 251 the mean mass-flux field $\rho \mathbf{u}$ and the turbulent diffusivity profile obtained from
 252 the turbulent viscosity ν_t of the flow. The mean mass-flux field field governs the
 253 advective transport of scalars in LEM3D, while the turbulent diffusivity governs
 254 the turbulent advection (stirring) by determining the rate at which turbulent eddy
 255 events occur. Both the mass-flux $\rho \mathbf{u}$ and the turbulent diffusivity typically vary in
 256 the spatial directions but are resolved only at the coarser length scale correspond-
 257 ing to the 3DCVs. The values of ν_t are fed to the centers of the control volumes,
 258 while face-normal components of \mathbf{u} are provided to the 3DCV faces.

Other model inputs to LEM3D include local (within the control volumes) val-
 ues for the integral length scale L_{int} and the Kolmogorov scale η , as well as a
 value for the scaling exponent p that governs the eddy-size dependence in the
 Kolmogorov inertial cascade range. The inputs are calculated from the k - ε model
 such that

$$\nu_t = C_\mu \frac{k^2}{\varepsilon}, \quad (5)$$

$$\eta = L_{\text{int}} \left(\frac{\nu_M}{\nu_t} \right)^{3/4}, \quad (6)$$

259 where $C_\mu = 0.09$ [28]. As in [29], the scaling exponent p is set equal to $4/3$.
 260 We here aim to demonstrate the LEM3D-Fluent coupling using a coarse steady-
 261 state RANS simulation in Fluent for which there is a one-to-one correspondence
 262 between the RANS grid cells and the 3DCVs. The LEM3D simulation domain
 263 is a cuboidal $45 \times 45 \times 84$ grid and thus here a sub-domain of the Fluent domain.

264 However, a Cartesian mesh is employed in the RANS simulation whose control
265 volumes coincide with the 3DCVs of the sub-domain. In this case, no interpolation
266 is needed and the values of the turbulent diffusivity and the face-normal mass-flux
267 components can be used as direct input to LEM3D. The input profiles are obtained
268 by user-defined functions (UDFs) in Fluent which format the data in line with the
269 proper input format for LEM3D. The LEM3D simulation is performed with the
270 conditions presented in Table 3. The advective time advancement Δt is calculated
271 through an inverse calculation setting the advective CFL number equal to 0.1.
272 Note, however, that the given approach and settings are done for simplicity and
273 that any RANS grid could be interpolated into a suitable mesh for LEM3D.

274 **3 Results and discussion**

275 The main objective of the present work has been to report on a new methodology
276 for modelling and simulation of reactive flows in which a 3D formulation of the
277 Linear Eddy Model LEM3D is used as a post-processing tool for an initial RANS
278 simulation. In this hybrid approach, LEM3D complements RANS with unsteadiness
279 and fine-scale resolution of scalar concentration profiles. The benefit of the
280 hybrid model, compared to a corresponding DNS, is the huge cost saving factor
281 of solving the reactive-diffusive equations on 1D domains, rather than in a full
282 3D computation. To leading order, the computational cost saving is estimated to
283 be $\sim 10^4$ for this particular application, based on a fine-scale resolution of about
284 300 LEM wafers in each coordinate direction within each 3DCV. To demonstrate
285 and fully challenge the RANS-LEM3D model, the hybrid model has here been
286 applied to the UC Berkeley vitiated co-flow burner. The results of the study are
287 presented in the following, with centerline scatter plots of various scalar quantities,
288 OH contour plots in the centerline symmetry plane, and axial profiles of scalars
289 along the centerline of the computational domain. The mixture fraction used in
290 the result section is computed using Bilgers formula [30] based on the elemental
291 mass fractions of the fuel and oxidizer.

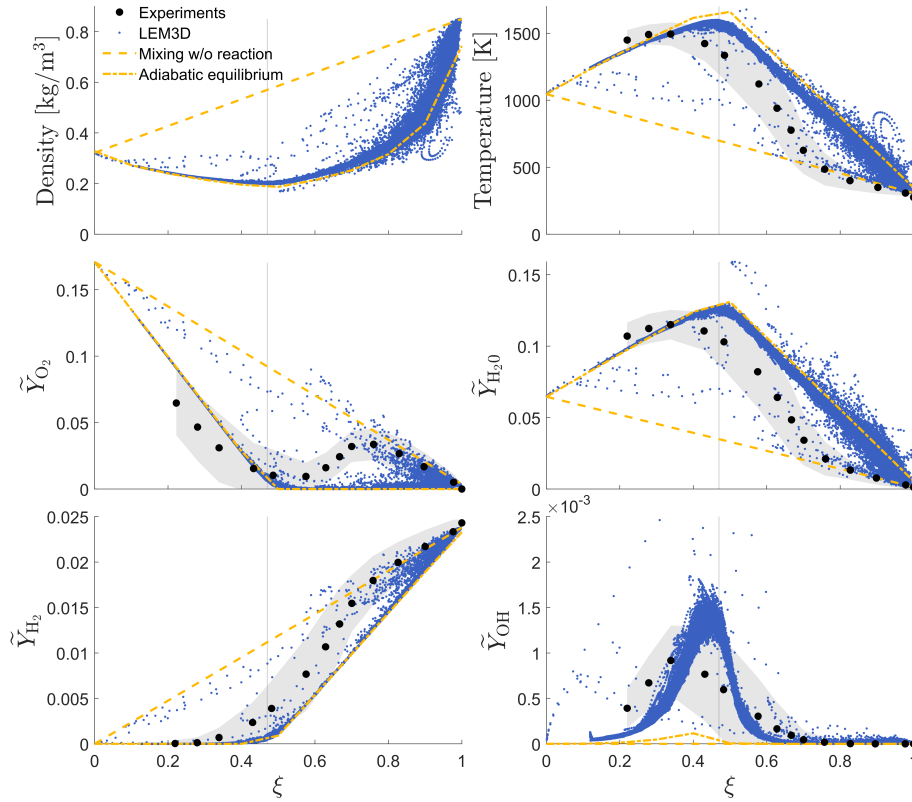


Fig. 3: Scatter plots on the centerline 1D LEM domain for various scalars versus the mixture fraction. The vertical line represents the stoichiometric mixture fraction ξ_{st} , while the gray-shaded areas represent the uncertainties of the experimental measurements [10], i.e., the variance of the scalars.

292 3.1 Scatter profiles

293 Figure 3 shows scatter plots of various scalar quantities versus the mixture frac-
 294 tion for the axial centerline LEM domain, together with experimental means and
 295 variances illustrated by the gray-shaded areas representing interpolated variance
 296 data taken from [10]. The dashed-dot-dashed curves represent the adiabatic equi-
 297 librium condition, computed with LOGEsoft [31] (and cross-checked with ANSYS
 298 Fluent), and the dashed curves are for mixing without reaction.

299 For each of the scatterplots, 41 samples are collected and plotted for the axial
 300 centerline domain, resulting in a total of 344400 points (the sum of centerline

LEM wafers sampled 41 times). The samples are collected every flow-through time after the flame has converged to a stable lift-off, and thus the scatters represent a collection of instantaneous states over the statistically steady sampling period. The scatters show reasonable agreement with the experimental curvatures, and capture both outliers as well as more typical states. There are in some cases tendencies of a large spread, which is likely due to the largest triplet maps. This, however, is a known artifact of the model for which the very large triplet maps in some instances create too sharp gradients [7], e.g., between the fuel jet and the surrounding oxygen stream.

In comparison with the experimental results we observe that the simulation results generally lie closer to the adiabatic equilibrium lines than the measurements. Further, both for hydrogen and oxygen we observe a split in the scatters for low values of ξ , which indicates the presence of both reacting and non-reacting wafers on the centerline.

3.2 Contour plots

The flame locations of the RANS and the subsequent LEM3D simulation are illustrated in Fig. 4 through OH contours. In the plots, only the RANS/3DCV cells for which \tilde{Y}_{OH} is larger than 600 ppm are shown. For LEM3D, 41 samples are collected over a time period corresponding to about 200 flow-through times.

Even though the flame stabilizes differently, both have a lift-off of approximately $1.4d$, based on our strict definition of lift-off height. However, by re-defining the lift-off as the first appearance of the continuous contour area for which $\tilde{Y}_{\text{OH}} > 600$ ppm, we get a lift-off of about $5.9d$ for LEM3D. Note that LEM3D gives a flame that is a bit radially displaced outwards compared to the RANS simulation. That is, for RANS the flame is located radially at around $r/d \approx 1$, while for LEM3D it is closer to $r/d \approx 2$. We further observe that the main burning rate upstream of $z/d \approx 20$, both for RANS and LEM3D, is radially bounded by $r/d \approx 4$.

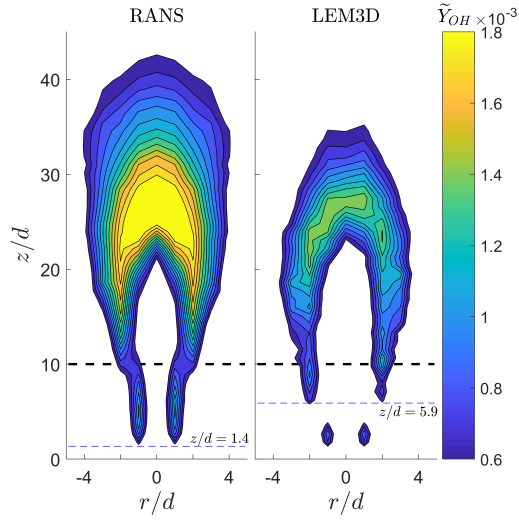


Fig. 4: Flame localization illustrated with OH contour plots of RANS versus LEM3D for the centerline symmetry plane. The black dashed line indicates the experimental lift-off $z/d = 10$, while the blue dashed lines show the computed continuous lift-off in either case.

329 3.3 Axial profiles

330 Axial profiles along the centerline for various scalars are shown in Fig. 5, together
 331 with RANS results and experimental data [9, 10]. There was no reported variance
 332 for the mixture fraction, hence no error bars are given in the ξ plot. From the
 333 mixture fraction plot, we observe that for $z/d \lesssim 25$ the co-flow fluid is reaching
 334 the centerline axial domain at a lower rate than indicated by RANS and the mea-
 335 surements. In general, however, the 3DCV-averaged curves are reasonably close
 336 to the data from Cabra et al. [9], except for the O_2 curve where LEM3D gives
 337 no initial peak as found in the experiment. A possible explanation for this is that
 338 the O_2 has been consumed and reacted to form H_2O in the radial domain $r/d = 2$
 339 in LEM3D. We observe that there is H_2O at the centerline but very little O_2
 340 upstream of $z/d \approx 25$, which indicates the lack of intrusion of unmixed co-flow
 341 fluid.

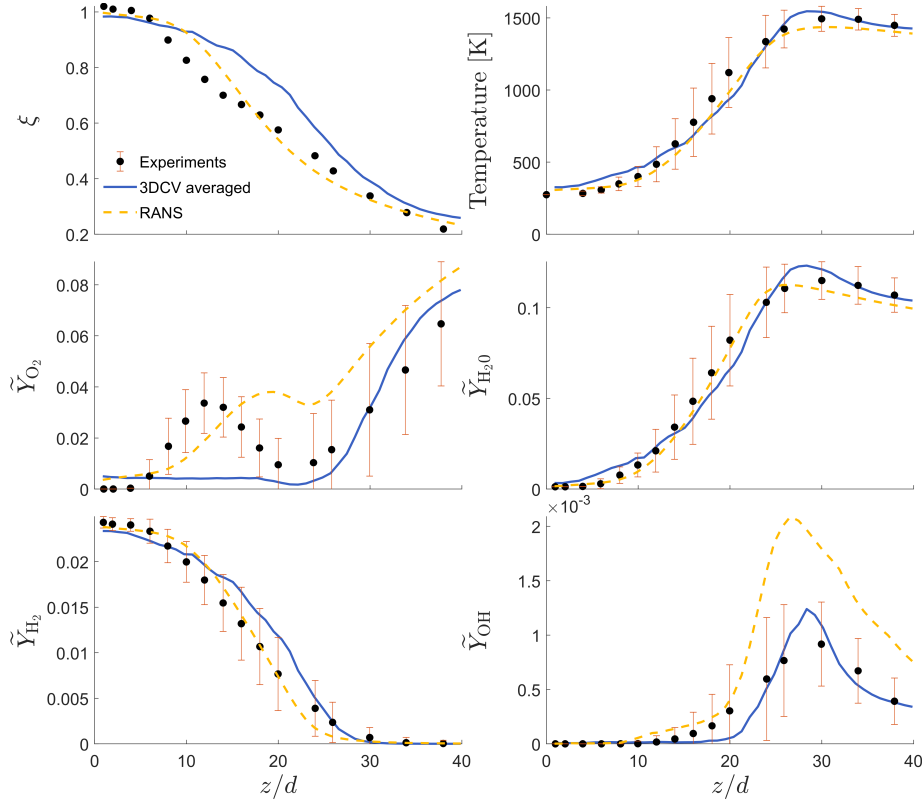


Fig. 5: Simulated axial profiles versus the measurements [9, 10] along the centerline. 3DCV averaged denotes the average value of all three LEM domains intersecting the centerline 3DCVs.

342 The presence of the initial O_2 peak in the measurements, and in RANS, is
 343 most likely either due to unmixed co-flow fluid reaching the centerline or to slow
 344 chemistry caused by the low temperature at the centerline. It is, however, reason-
 345 able to assume that the unreacted O_2 is due to incomplete mixing rather than to
 346 slow chemistry. Otherwise, since LEM3D is running the same chemistry as RANS,
 347 unreacted O_2 should also have shown up at the centerline in that simulation. This
 348 is supported by the flame stabilization plots in Fig. 4, which indicate that it would
 349 take longer for the OH to diffuse to the centerline for LEM3D. Hence, very little
 350 OH reaches the centerline before $z/d \approx 20$ since it reacts to form H_2O on the way.

351 In LEM3D, the first appearance of OH at the centerline is seen at $z/d \approx 20$.
352 This is slightly later than indicated by the measurements of Cabra et al. [9] and
353 by RANS, and is in agreement with the contour profiles of Fig. 4. Hence, this
354 is where the chemical reactions start at the centerline and we see an increase
355 in the gradients of both the temperature and the H₂O 3DCV-averaged curves
356 downstream of $z/d = 20$.

357 4 Conclusions

358 The present paper reports on a new methodology for modeling and simulation
359 of reactive flows in which LEM3D is used as a post-processing tool for an initial
360 RANS simulation. In this hybrid modeling approach, LEM3D complements RANS
361 with unsteadiness and small-scale resolution of scalar concentration profiles.

362 To demonstrate the RANS-LEM3D approach, the hybrid model is here ap-
363 plied to the UC Berkeley vitiated co-flow burner first presented by Cabra et al.
364 [9, 10]. From the RANS output, LEM3D in general provides spatial and temporal
365 information in good agreement with the experimental measurements. PDF trans-
366 port methods are known to produce similar scatter plots as shown in Fig. 3, but
367 ODT, which subsumes the capabilities of LEM, has been shown to provide better
368 agreement with detailed DNS results than obtained using other models [32].

369 The turbulent lifted N₂-diluted hydrogen jet flame is challenging due to the
370 high sensitivity of its lift-off height, hysteresis effects, and competing flame stabi-
371 lization mechanisms [13, 27]. Here, a RANS solution based on the same numerical
372 scheme and boundary conditions as employed by Myhrvold et al. [14] was used
373 as model input to LEM3D. With the given Fluent default values of the standard
374 k - ε model, and the modification of the parameter $C_{2\varepsilon}$ to correct for the spreading
375 rate, the RANS simulation provided a close-to-attached flame.

376 The centerline axial profiles of scalars are, with the exception of the O₂ curve,
377 generally in good agreement with the measurements by Cabra et al. [9]. The in-
378 capability of capturing the initial peak of the O₂ curve may be due to a known

379 model artifact in LEM that causes near-field discrepancies resulting from the in-
380 stantaneous nature of the eddy events [2, 22]. However, it may also be due to
381 inaccuracies in the input flow field due to the coarse RANS grid resolution or the
382 fact that the initial RANS simulation provided a close-to-attached flame.

383 It has been noted that the flame configuration studied here is especially chal-
384 lenging for RANS-based modeling owing to the strong dependence of the results
385 on the specification of RANS inputs. In such a situation, RANS-based combus-
386 tion modeling is more useful for sensitivity analysis than for point prediction. In
387 addition to the results presented here, numerous excursion cases have been run
388 involving adjustment of both RANS and LEM3D parameters as well as variants
389 of the LEM3D formulation. They indicate that agreement of particular outputs
390 with the measurements improve or decline on a case-by-case basis. Nevertheless,
391 the chosen flame configuration involves a degree of complexity such that the addi-
392 tional chemical detail provided by LEM3D, such as various scatter plots that are
393 shown and statistics that are potentially extractable from them, could be useful
394 for diagnosing the implications of particular RANS outcomes and more generally
395 for sensitivity studies focused on identification of trends. This is the intended role
396 of LEM3D post-processing of RANS combustion solutions. In the current work,
397 the average LEM3D flame location given by the OH-contours does not coincide
398 with the RANS flame location. Post-processing tools should in general coincide
399 with the input on average, and improvements in this regard will be addressed in
400 future work.

401 To conclude, the hybrid RANS-LEM3D methodology has here been demon-
402 strated by application to the UC Berkeley vitiated co-flow burner. As a post-
403 processing tool to RANS, LEM3D can provide additional scalar statistics and
404 more detailed information on the flame structure and the small-scale mixing reac-
405 tive flows. The advantage of the RANS-LEM3D model, compared to a DNS with
406 a corresponding fine-scale resolution, is that the hybrid model represents a com-

407 putationally cost-efficient tool that can predict certain flame characteristics not
408 available from RANS alone.

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412 **Compliance with Ethical Standards**

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414 **Conflict of interests** The authors declare that they have no conflict of interest.

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